

Description

Integral Locking Coupler

CROSS REFERENCE TO RELATED APPLICATIONS

[0001] This application claims the benefit of United States Provisional Patent Application No. 60/320,230, filed May 28, 2003, the entire disclosure of which is hereby incorporated by reference.

BACKGROUND OF INVENTION

[0002] The present invention relates to an improved trailer coupler and more particularly to a trailer coupler to provides easy engagement and disengagement from a receiver and further includes a locking mechanism.

[0003] Typical vehicle hitch mechanism employ a mount, a hitch pin and a lock. When engaging the standard hitch mechanism, all three of the pieces, namely the mount, the hitch pin, and the lock, must be aligned and properly placed. Each must be assembled separately, thus taking substantial time and effort. This can be especially troublesome in inclement weather and on hitches that are located close to

the pavement or shrouded by bumpers. The mount or coupler is placed in the receiver, properly aligned and then secured with a hitch pin. A lock is then applied to the hitch pin to secure the mounting assembly.

[0004] Prior art assemblies have attempted to combine certain of these features, however have not provided a fully-functioning locking coupler. For example, in some references a locking mechanism is not employed, thereby requiring a separate locking component. These references generally include spring-biased bolts that move in and out to engage the coupler. In some references, the bolts must be manually forced inward in order to retract the bolts and allow for insertion into the receiver. In other references, the bolts are moved in and out through engagement of a surface that has different dimension depending on the placement of the surface. For example, a blocker may be used, wherein the blocker has two different dimensions. The first dimension is larger and is positioned for maintaining the bolts in the outward position. The second dimension is smaller and is positioned for allowing the bolts to be in the inward, retracted position. This type of coupler require a large piece that must be moveable within the coupler. In some references, complex

drive gears and gear racks are employed to move the engagement surface. In embodiments such as the use of a blocker or a drive gear and slidable engagement surface, the mechanism is not coupled to a lock cylinder and does not allow for optimum mechanical advantage. As such, there exists a need for an improved coupler that not only allows integration of the mount, hitch pin and lock , but also operates in a manner such as to provide optimal mechanical advantage in movement of the pieces and is cost efficient to produce.

SUMMARY OF INVENTION

[0005] The present invention relates to an integral locking coupler. The integral locking coupler includes a locking mechanism that is coupled to a set of engagement pins. The movement of the locking mechanism translates to the movement of the engagement pins to and from an extended position and a retracted position. When in the retracted position, the coupler can be inserted or removed from a receiver. When in the extended position, the engagement pins penetrate through holes in the receiver and secure the coupler to the receiver. The locking mechanism ensures that the engagement pins are maintained in the extended position and can not be manipulated to

remove the coupler without unlocking the locking mechanism.

BRIEF DESCRIPTION OF DRAWINGS

- [0006] The present invention will be more fully understood by reference to the following detailed description of the invention and the accompanying drawings. The drawings represent exemplary embodiments of the present invention and are included for illustrative purposes in order to facilitate understanding of the description. Other embodiments of the present invention contemplated by the description are included within this application to the extent they fall within the scope of the claims attached hereto.
- [0007] Figure 1 is a perspective view of a integral locking ball mount of the present invention as applied to a towing hitch bar.
- [0008] Figure 2 is a side plan view of the integral locking ball mount shown in Figure 1.
- [0009] Figure 3 is an exploded view of the integral locking ball mount shown in Figure 1.
- [0010] Figure 4 is an exploded view of the internal components of the integral locking ball mount shown in Figure 1.
- [0011] Figure 5 is a perspective assembly drawing of the internal components shown in Figure 4.

- [0012] Figure 6 illustrates an alternative embodiment of the integral locking ball mount of the present invention.
- [0013] Figure 7 illustrates an alternative embodiment of the integral locking ball mount of the present invention incorporating a rotating cam and shaft.
- [0014] Figure 8 illustrates an another alternative embodiment of the integral locking ball mount of the present invention incorporating a rotating cam and shaft.
- [0015] Figure 9 illustrates the locking pin subassembly of the rotating cam and shaft embodiments.
- [0016] Figure 10 illustrates the rotating cam of Figure 9.
- [0017] Figure 11 illustrates an alternative embodiment of the invention, wherein the integral coupling is employed in combination with a cargo carrier.

DETAILED DESCRIPTION

- [0018] The present invention is a integral locking ball mount, or coupler, *10* that performs the functions of a standard ball mount, hitch pin and lock. The coupler *10* include a lock cylinder *15* that is coupled to an actuation mechanism that uses optimal mechanical advantage to move a set of engagement pins *20* to and from an extended engagement position and a retracted disengagement position. This al-

allows for the coupler 10 to easily installed and removed from a trailer receiver with minimal effort, while still maintaining maximum lock security of the coupler.

[0019] The coupler 10 includes a coupler housing 12 with an internal housing cavity 14 and two apertures 16 which align with the engagement pins 20. When the engagement pins 20 are in the extended position, the engagements pins 20 extend outward through the apertures 16 and beyond the coupler housing 12. When the engagement pins 20 are in the retracted position, the engagement pins 20 are drawn inward through apertures 16 and are either flush with the coupler housing 12 or are moved entirely or substantially within the housing cavity 14.

[0020] The integral locking ball mount 10 generally includes a ball mount plate 40, lock housing 44, lock cylinder, or other lock mechanism, 15 and locking pin subassembly 45. The locking pin subassembly 45, shown in Figure 4, generally includes two engagement pins 20, two sliding pins 48, a biasing means, such as a spring, 50 and a housing 52. The housing 52 as shown include a main portion 53 and two slide plates 54. The side plates 54 include apertures 55 through which the engagement pins 20 extend. The biasing means 50 applies force on the engagement

pins 20 forcing them outwards into the engagement position.

[0021] The cylinder 15 can be any conventional lock cylinder and is generally disposed within lock cylinder housing 60. The cylinder 15 engages cam 65 which engages lever 70. The cam 65 includes a protrusion 67 that engages slot 72 of the lever 70. In general, any mechanical coupling of the cylinder 15 to the lever 70 can be used provided that the rotation of the cylinder moves the lever in the horizontal plane, as shown in Figure 4. As shown in Figure 4, the slot 72 is located on a tab 73 that extends outward away from the main portion 74 of the lever. As the cylinder 15 is rotated to the unlocked position, generally a 180 degree rotation, the protrusion 67 rotates pulling the tab 73 and the lever 70 away from the engagement pins 20. This allows the engagement pins 20 to be retracted and the coupler 10 to be inserted or removed from the receiver, as explain in further detail below.

[0022] Lever 70 further includes pin slots 77 which engage sliding pins 48. Pin slots 77 are generally c-shaped, wherein the bottom of the c-shape 77b is angled inward toward the center of the lever 70. As such, when the sliding pins 48 are moved along the pin slots 77 the sliding pins 48 not

only travel axially, but also travel radially inward toward the center of the of coupler 10. Since the sliding pins 48 are coupled to the engagement pins 20, the axial movement of the sliding pins 48 cause the engagement pins 20 to extend and retract. This movement of the engagement pins 20 allows the coupler 10 to be inserted and removed from a receiver. Optionally, the lever 70 may include a blocker 78 which extends from the lever 70 to be in between the engagement pins 20 when in the extended position. The blocker 78 provides additional security against attempts to manually retract the engagement pins 20. The blocker 78 can be any shape or configuration provided it provides additional security to prevent the movement of the engagement pins to the retracted position when the coupler is in the locked position.

[0023] To install the coupler 10 shown in Figures 1–5, a key (not shown) is inserted into the lock cylinder 15 and rotated 180 degrees to unlock the cylinder. The rotation of the cylinder 15 slides the lever 70 axially through engagement with cam 65. The axial movement of the lever 70 moves the sliding pins 48 along pin slots 77 and moves the blocker 78 away from the engagement pins 20. The engagement pins 20 are thus retracted into the coupler

housing cavity 14 and thus do not extend beyond the coupler housing 12. So positioned, the coupler 10 can be inserted into a receiver 80 in conventional fashion. The engagement pins 20 are aligned with apertures 82 in the receiver and then the cylinder 15 is rotated back to the locked position. In doing so, the lever 70 slides back to the engagement position, wherein the slide pins 48 move to the radially outward portion of the pin slots 77. The movement of the sliding pins 48 move the engagement pins 20 to the extended position, thereby penetrating through the coupler housing 12 and through the holes 82 in the receiver 80. If a blocker 78 is used, the blocker will be positioned between the engagement pins 20 when in the extended position to further prevent inward movement of the engagement pins. The spring 50 biases the engagement pins 20 outward towards the extended position. It should be appreciated that the spring or biasing means 50 is not required to move the engagement pins 20 to or from the engagement position, but the inclusion of such biasing means provides additional security in maintaining the engagement pins in the extended position.

[0024] To remove the coupler shown in Figures 1–5, the key is inserted into the lock cylinder 15 and rotated 180 degrees

to the unlocked position. The rotation of the lock cylinder 15 moves the lever 70 and draws the engagement pins inward into the coupler cavity 14 and out of engagement with the receiver apertures 82. The coupler 10 can then be removed from the receiver 80.

[0025] Figure 6 illustrates a different lever mechanism. The cylinder 15 in Figure 6 engages blocker 90 and moves it up and down, as shown in Figure 6, to engage and disengage the lever 70. Once disengaged, the lever 70 can be pulled axially outward by manipulation of lever portion 92. The movement of the lever 70 then moves the engagement pins inward or outward as described above.

[0026] Figures 7 and 8 illustrate another embodiment of the integral ball mount of the present invention. Figures 7 and 8 differ merely in the location of the lock cylinder 15. The lever of the prior embodiment is replaced with a shaft 100 coupling the lock cylinder 15 to the engagement pins 20. Rotation of the lock cylinder 15 allows for rotation of the shaft 100, which in turn rotates cam 105, as best shown in Figure 10. The cam 105 includes two sliding pins recesses 108 which start towards the outer periphery 109 of the cam 105 and gradually tapers inward toward the center of the cam 105. The sliding pins 48 are coupled to the engage-

ment pins *20* such that when cam *105* is rotated, the sliding pins *48* move circumferentially and radially inward toward the center the cam *105*, thereby drawing the engagement pins *20* into the coupler cavity *12*. So positioned, the coupler can be inserted or removed from a receiver. Rotation of the shaft *100* in the other direction, will rotate the cam *105* back and move the engagement pins *20* to the extended position. Although not shown in the illustrative examples, one skilled in the art should appreciate that the cylinder and shaft can be mechanically coupled such that rotation of the cylinder would not only unlock the cylinder, but also translate the rotation to the shaft.

[0027] Figure 11 illustrates an alternative embodiment of the invention. Figure 11 illustrates the coupling mechanism applied to a cargo carrier *110*. One skilled in the art should appreciate that the coupling mechanism described herein can be applied not only to a ball mount, but also to cargo carriers, bike carriers, and the like. Such embodiments are meant to be included within the scope of this application.

[0028] It should be appreciated that the locking mechanism *15* can be a variety of mechanisms that provide the end user with a means for locking the integral locking ball mount. For example, the locking mechanism can be an inter-

changable wafer lock cylinder, non-interchangable wafer lock cylinder, pin tumbler cylinder, a push-button lock cylinder, or a tubular lock cylinder. In one embodiment the locking mechanism *15* is located along a side wall portion *120* of the coupler housing *12*. In another embodiments, the locking mechanism *15* is located at the end of the coupler housing *12* or apart of the shaft of lever coupling the lock mechanism to the engagement pins.

[0029] The detailed description has been provided for clearness of understanding of the present invention and is not intended to unnecessarily limit the scope of the invention. Modifications will be obvious to those of ordinary skill in the art, and such modifications are intended to be included as part of this invention to the extent they fall within the scope of the claims attached hereto.